

Problem 1) A) 1)

$$X_d = Z X_d + \frac{1}{2} C_0 \quad (Z = 5)$$

$$X_d (1 - Z) = \frac{1}{2} C_0$$

$$C_0 = 2 X_d (1 - Z)$$

$$Z = 0.5 \Rightarrow C_0 = X_d,$$

$$Z = 0.9 \Rightarrow C_0 = 0.2 X_d$$

2) from above, we know in a

Patient with $Z = 0.9$ $X_d = 5 C_0$.

Lets call $Z = 0.5$ Patient's dose

$$C_0 = X_d'$$

If we give $C_0 = X_d'$ to $Z = 0.9$ Patient, This Patient has Blood Stream Concentration of $S X_d'$. $S X$ the Expected concentration (Not Good).

B) 1) no oscillation means $\lambda > 0$.

Substitute $C[n-1]$ into model.

$$X[n] = Z X[n-1] + \frac{1}{2} K_p (X_d - X[n-1])$$

$$X[n] = \frac{1}{2} K_p X_d + \underbrace{\left(Z - \frac{1}{2} K_p \right)}_{\lambda} X[n-1]$$

first order model, λ is above

$$\lambda = Z - \frac{1}{2} K_p > 0$$

$$K_p < 2 \cdot Z$$

Z can range between 0.5 & 0.9,

Lets be safe and assume $Z = 0.5$

$$K_p = 1$$

2) In steady state, $X[n] = X[n-1] = X_{ss}$

$$X_{ss} = \frac{1}{2} K_p X_d + \left(Z - \frac{1}{2} K_p \right) X_{ss}$$

$$X_{ss} \left(1 - Z + \frac{1}{2} K_p \right) = \frac{K_p}{2} X_d$$

$$\frac{X_{ss}}{X_d} = \frac{K_p/2}{1 - Z + \frac{K_p}{2}}$$

$$K_P=1 \Rightarrow \frac{X_{SS}}{X_d} = \frac{1/2}{\frac{3}{2}-z} = \frac{1}{3-2z}$$

$$z=0.5 \Rightarrow \frac{X_{SS}}{X_d} = \frac{1}{2}$$

$$z=0.9 \Rightarrow \frac{X_{SS}}{X_d} = 0.833$$

With P control, our X_{SS} ranges between 50% and 83% of our desired concentration, much less variable than open loop's 20% - 100% range. This is better.

c) 1)

$$C[n-1] = K_p (X_d - X[n-1]) + K_i S[n-1]$$

$$X[n] = \frac{1}{2} K_p X_d + \left(Z - \frac{1}{2} K_p \right) X[n-1] + \frac{1}{2} K_i S[n-1]$$

$$S[n] = S[n-1] + X_d - X[n-1]$$

$$\vec{B} = \begin{bmatrix} \frac{K_i}{2} \\ 1 \end{bmatrix}$$

$$\bar{A} = \begin{bmatrix} Z - K_p/2 & \frac{K_i}{2} \\ -1 & 1 \end{bmatrix}$$

2) How do \bar{A} , \vec{B} , and X_d affect our Steady State \vec{X}_{ss} ?

If all of our λ 's have magnitude < 1 , then $\vec{X}[n] = \vec{X}[n-1] = \vec{X}_{ss}$

$$\vec{X}_{ss} = \bar{A} \vec{X}_{ss} + \vec{B} \cdot X_d$$

$$\vec{X}_{ss} - \bar{A} \vec{X}_{ss} = \vec{B} \cdot X_d$$

$$(\mathbf{I} - \bar{A}) \vec{X}_{ss} = \vec{B} \cdot X_d$$

$$\vec{X}_{ss} = (\mathbf{I} - \bar{A})^{-1} \vec{B} \cdot X_d$$

If we know X_d and $(\mathbf{I} - \bar{A})^{-1} \vec{B}$, we can find \vec{X}_{ss} .

OK, what does $(I - \bar{A})^{-1} \bar{B}$ tell us about

The relationship between X_d and \vec{X}_{ss} ?

Matlab says $(I - \bar{A})^{-1} \bar{B}$ is: $\begin{bmatrix} 1 \\ \frac{2(1-z)}{k_i} \end{bmatrix}$

$$\vec{X}_{ss} = \begin{bmatrix} X[\infty] \\ s[\infty] \end{bmatrix} = \begin{bmatrix} 1 \\ \frac{2(1-z)}{k_i} \end{bmatrix} X_d$$

$$X[\infty] = X_d$$

3) I would use Matlab to plot how the natural frequencies move as I change K_p and K_i . I would need to make sure all natural frequencies lie within the unit circle to ensure stability.

$$D) \quad 1) \\ \bar{E} = \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix}$$

$$\tilde{A}_{new} = \begin{bmatrix} z - k_p/2 & k_i/2 \\ 0 & 1 \end{bmatrix}$$

$$\tilde{B} = \begin{bmatrix} k_p/2 \\ 1 \end{bmatrix}$$

D) 2) From Matlab:

$$\tilde{A} = \bar{E}^{-1} \bar{A}_{new} = \begin{bmatrix} z - \frac{k_p}{2} & \frac{k_i}{2} \\ \frac{k_p}{2} - z & 1 - \frac{k_i}{2} \end{bmatrix}$$

$$\tilde{B} = \bar{E}^{-1} \bar{B} = \begin{bmatrix} k_p/2 \\ 1 - \frac{k_p}{2} \end{bmatrix}$$

$$(\mathbf{I} - \tilde{A})^{-1} \tilde{B} = \begin{bmatrix} 1 \\ \frac{2(1-z)}{k_i} \end{bmatrix}$$

Same as Part C 2

Problem 2)

Our \tilde{B}_{in} should have a non zero component only in the states directly affected by error $[n]$. ($sum[n]$ and $\alpha[n]$)

$$\alpha[n] = (1 + \Delta T B) \alpha[n-1] - \Delta T B \gamma_a \left(K_p e[n-1] + K_d del[n-1] + K_i sum[n-1] \right)$$

$$sum[n] = \Delta T e[n] + sum[n-1]$$

In $\alpha[n]$, The coefficients with $e[n]$ are

The coefficients in \tilde{B}_{in} (Because $e[n] = \theta_d - \theta[n]$)

The same logic follows for $sum[n]$.

$$B_{in} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ -\beta \Delta T \gamma_n K_p \\ \Delta T \end{bmatrix}$$

To check our work, we can make sure that in steady state $\theta[n] = \theta[n-1] = \theta_d$

We use $(\bar{I} - \bar{A})^{-1} \bar{B}_{in}$ to find steady state vector.

$$\begin{bmatrix} \theta[\infty] \\ \theta[n-1] \\ w[\infty] \\ \alpha[\infty] \\ s_{um}[\infty] \end{bmatrix} = (\bar{I} - \bar{A})^{-1} \bar{B}_{in} \theta_d$$

$$\begin{bmatrix} \theta[\infty] \\ \theta[n-1] \\ w[\infty] \\ \alpha[\infty] \\ s_{um}[\infty] \end{bmatrix} = \begin{bmatrix} 1 \\ 1 \\ 0 \\ 0 \\ 0 \end{bmatrix} \theta_d$$

We see $\theta[n] = \theta[n-1] = \theta_d$ in steady state $[n = \infty]$.

To find \vec{B}_{dist} , think about how we can "inject" a disturbance force into our system.

On first thought, we might think to inject the disturbance force into α as α represents angular acceleration. In reality, α represents acceleration due to the force coming from the propellers.

With an undisturbed system, these two quantities are the same. For this problem, this is not true.

ω is the angular velocity of our arm. All forces contribute to it. It makes the most sense to "inject" our disturbance force into the $\omega[n]$ state.

$$\vec{B}_{dist} = \begin{bmatrix} 0 \\ 0 \\ 1 \\ 0 \\ 0 \end{bmatrix}$$

To be safe, we can make sure a disturbing force will lead to a zero error in θ_a and ω_a .

$$\begin{bmatrix} \theta [\infty] \\ \dot{\theta} [\infty] \\ w [\infty] \\ \alpha [\infty] \\ s_{um} [\infty] \end{bmatrix} = (\bar{I} - \bar{A})^{-1} \bar{B} \downarrow_{dist} u_{dist}$$

$$\begin{bmatrix} \theta [\infty] \\ \dot{\theta} [\infty] \\ w [\infty] \\ \alpha [\infty] \\ s_{um} [\infty] \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ -1/\Delta T \\ -1/\delta_a k_i \Delta T \end{bmatrix} u_{dist}$$

We see that Both θ and w go to zero from a disturbance force. α does not go to zero because the Propeller may need to generate a force to counteract the steady state disturbance force.