6.3100: Dynamic System Modeling and Control Design

**DT** Poles and Stability

September 30, 2024

### Last Time

We introduced an **operator** representation for discrete time systems. Example: **robotic steering** 

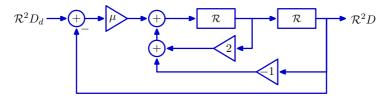
$$d[n] = d[n-1] + V\Delta T\theta[n-1] \qquad D = \mathcal{R}D + V\Delta T\mathcal{R}\Theta$$
  

$$\theta[n] = \theta[n-1] + \Delta T\omega[n-1] \qquad \Theta = \mathcal{R}\Theta + \Delta T\mathcal{R}\Omega$$
  

$$\omega[n] = \gamma u[n] \qquad \Omega = \gamma U$$
  

$$u[n] = K_p(d_d[n] - d[n]) \qquad U = K_p(D_d - D)$$

The operator representation (with  $\mathcal{R}$  representing **right-shift**) retains the structure of the underlying difference equations, and allows us to manipulate and simplify difference equations using **polynomials** of  $\mathcal{R}$ .



#### Last Time

By interpreting  $\mathcal{R}$  as **delay** we get a description in the time domain.

Time domain: step-by-step calculation of samples:

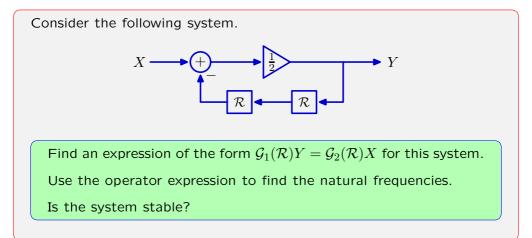
Interpreting  $\mathcal{R}$  as gain  $(\lambda^{-1})$  provides a description in the frequency domain. **Frequency domain**: constraints on the structure of the output **signal**:

$$\lambda^{n} = 2\lambda^{n-1} - \lambda^{n-2} - (\Delta T)^{2} V K_{p} \gamma \lambda^{n-2}$$

$$0 \longrightarrow + \lambda^{n} \lambda^{-1} \lambda^{n-1} \lambda^{-1} \lambda^{n-2}$$

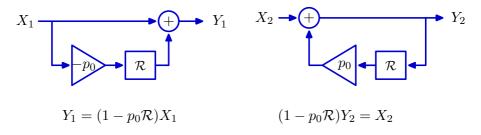
$$+ \lambda^{n-2} \lambda^{n-2}$$

## **Check Yourself**



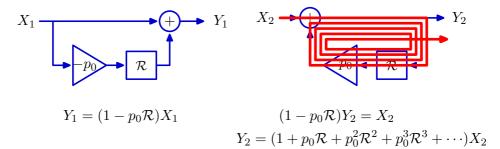
### Last Time: Analyzing Systems with Polynomials

Compare operator descriptions of these feedback and feedforward systems:



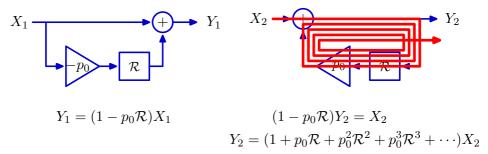
#### **Transient and Persistent Responses**

Compare operator descriptions of these feedback and feedforward systems:



### **Reciprocal Relations**

Compare operator descriptions of these feedback and feedforward systems:



Substitute  $X_2$  from the first equation into the second:

$$Y_2 = (1 + p_0 \mathcal{R} + p_0^2 \mathcal{R}^2 + p_0^3 \mathcal{R}^3 + \cdots)(1 - p_0 \mathcal{R})Y_2$$

and therefore

$$(1 + p_0 \mathcal{R} + p_0^2 \mathcal{R}^2 + p_0^3 \mathcal{R}^3 + \cdots)(1 - p_0 \mathcal{R}) = 1$$

The two factors  $1+p_0\mathcal{R}+p_0^2\mathcal{R}^2+p_0^3\mathcal{R}^3+\cdots$  and  $1-p_0\mathcal{R}$  must be **reciprocals**. We can think of the operator representation of this feedback system as

$$\mathcal{H}(\mathcal{R}) = \frac{1}{1 - p_0 \mathcal{R}} = 1 + p_0 \mathcal{R} + p_0^2 \mathcal{R}^2 + p_0^3 \mathcal{R}^3 + p_0^4 \mathcal{R}^4 + \cdots$$

### **Polynomial Interpretation of Reciprocals**

The reciprocal relation between the two representations

$$\mathcal{H}(\mathcal{R}) = \frac{1}{1 - p_0 \mathcal{R}} = 1 + p_0 \mathcal{R} + p_0^2 \mathcal{R}^2 + p_0^3 \mathcal{R}^3 + p_0^4 \mathcal{R}^4 + \cdots$$

also follows from polynomial division.

$$1 + p_0 \mathcal{R} + p_0^2 \mathcal{R}^2 + p_0^3 \mathcal{R}^3 + \cdots$$

$$1 - p_0 \mathcal{R} \boxed{1}$$

$$\frac{1 - p_0 \mathcal{R}}{p_0 \mathcal{R}}$$

$$\frac{p_0 \mathcal{R} - p_0^2 \mathcal{R}^2}{p_0^2 \mathcal{R}^2}$$

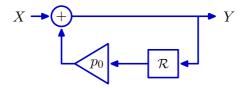
$$\frac{p_0^2 \mathcal{R}^2 - p_0^3 \mathcal{R}^3}{p_0^3 \mathcal{R}^3}$$

$$\frac{p_0^3 \mathcal{R}^3 - p_0^4 \mathcal{R}^4}{\cdots}$$

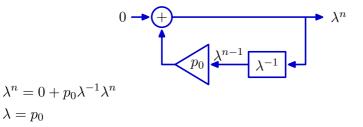
The normal rules of polynomial algebra apply to system operators.

## Using Operator Expressions in the Frequency Domain

Operators also simplify thinking in the frequency domain. Find the natural frequency of this system.



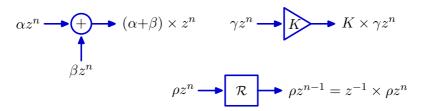
The natural frequency has the form  $Y = \lambda^n$  when the input X = 0.



The natural frequency  $\lambda$  is equal to  $p_0$ . This is a special case of a more general frequency result based on complex geometrics.

### **Geometric Signals**

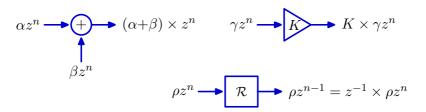
When the inputs to adders, gains, and delays are proportional to  $z^n$ , their outputs are also proportional to  $z^n$ .



If the output of a system is a scaled multiple of its input, we say that the input signal is an **eigenfunction** of the system.

## **Geometric Signals**

When the inputs to adders, gains, and delays are proportional to  $z^n$ , their outputs are also proportional to  $z^n$ .



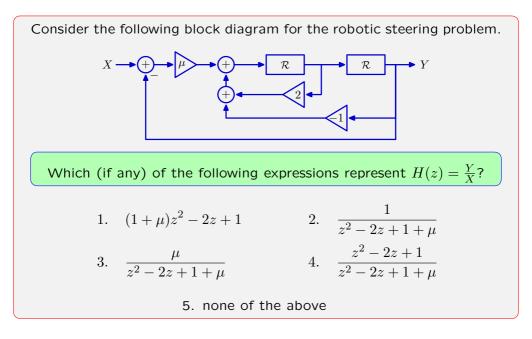
Similarly if the input to any combination of adders, gains, and delays is proportional to  $z^n$ , then the output is also proportional to  $z^n$ .

To find the constant of proportionality, simply substitute  $\frac{1}{z}$  for  $\mathcal{R}$  in the corresponding operator expression:

$$H(z) = \mathcal{H}(\mathcal{R})\Big|_{\mathcal{R} \to \frac{1}{z}}$$

H(z) is called the system function or transfer function.

### **Check Yourself**



### **Black's Equation**

More generally, let  $\mathcal{F}(\mathcal{R})$  represent the forward path and  $\mathcal{G}(\mathcal{R})$  represent the feedback path for a feedback system.

$$X \xrightarrow{E} \mathcal{F}(\mathcal{R}) \xrightarrow{E} \mathcal{F}(\mathcal{R}) \xrightarrow{F} Y$$

$$\mathcal{G}(\mathcal{R})$$

$$Y = \mathcal{F}(\mathcal{R})E = \mathcal{F}(\mathcal{R})\left(X + \mathcal{G}(\mathcal{R})Y\right) = \mathcal{F}(\mathcal{R})X + \mathcal{F}(\mathcal{R})\mathcal{G}(\mathcal{R})Y$$

$$\left(1 - \mathcal{F}(\mathcal{R})\mathcal{G}(\mathcal{R})\right)Y = \mathcal{F}(\mathcal{R})X$$

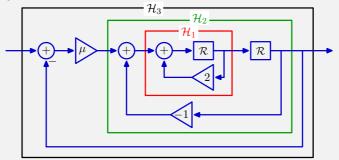
The transformation from X to Y is given by the operator expression

$$\mathcal{H}(\mathcal{R}) = \frac{Y}{X} = \frac{\mathcal{F}(\mathcal{R})}{1 - \mathcal{F}(\mathcal{R})\mathcal{G}(\mathcal{R})}$$

This equation is known as **Black's equation**.

## **Check Yourself**

Let  $\mathcal{H}_1$  and  $\mathcal{H}_2$  represent subsystems within the robotic steering block diagram  $\mathcal{H}_3$ .



How many of the following expressions are true?

$$\mathcal{H}_1 = \frac{\mathcal{R}}{1 - 2\mathcal{R}} \qquad \mathcal{H}_2 = \frac{\mathcal{H}_1 \mathcal{R}}{1 + \mathcal{H}_1 \mathcal{R}} \qquad \mathcal{H}_2 = \frac{\mathcal{R}^2}{1 - 2\mathcal{R} + \mathcal{R}^2}$$
$$\mathcal{H}_3 = \frac{\mu \mathcal{H}_2}{1 + \mu \mathcal{H}_2} \qquad \mathcal{H}_3 = \frac{\mu \mathcal{R}^2}{1 - 2\mathcal{R} + (1 + \mu)\mathcal{R}^2}$$

#### Modularity

If a feed-forward system contains only adders, gains, and delays, then it's system function can be expressed as a polynomial in  $\mathcal{R}$ .

$$\mathcal{H}(\mathcal{R}) = b_0 + b_1 \mathcal{R} + b_2 \mathcal{R}^2 + b_3 \mathcal{R}^3 + \cdots$$

Responses of such systems are transient in the sense that their outputs go to zero no later than N time steps after their input goes to zero, where N is the degree of  $\mathcal{H}(\mathcal{R})$ .

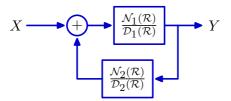
If both the forward and feedback paths through a system with feedback can be represented as polynomials in  $\mathcal{R}$ , then the system function can be expressed as a **rational polynomial** in  $\mathcal{R}$ .

$$\mathcal{H}(\mathcal{R}) = \frac{b_0 + b_1 \mathcal{R} + b_2 \mathcal{R}^2 + b_3 \mathcal{R}^3 + \cdots}{1 + a_1 \mathcal{R} + a_2 \mathcal{R}^2 + a_3 \mathcal{R}^3 + \cdots}$$

What happens if a system contains a feedback system within a feedback system?

#### Modularity

If the forward path through a feedback system contains feedback, then the forward path can be represented by a rational polynomial  $\mathcal{N}_1(\mathcal{R})/\mathcal{D}_1(\mathcal{R})$ . If the feedback path through a feedback system contains feedback, then the feedback path can be represented by a rational polynomial  $\mathcal{N}_2(\mathcal{R})/\mathcal{D}_2(\mathcal{R})$ .



We can apply Black's formula to find the resulting system function:

$$\frac{Y}{X} = \frac{\frac{N_1(\mathcal{R})}{D_1(\mathcal{R})}}{1 - \frac{N_1(\mathcal{R})}{D_1(\mathcal{R})}\frac{N_2(\mathcal{R})}{D_2(\mathcal{R})}} = \frac{\mathcal{N}_1(\mathcal{R})\mathcal{D}_2(\mathcal{R})}{\mathcal{D}_1(\mathcal{R})\mathcal{D}_2(\mathcal{R}) - \mathcal{N}_1(\mathcal{R})\mathcal{N}_2(\mathcal{R})}$$

Since the product of polynomials is polynomial, it follows that the overall system function is a rational polynomial.

#### **Partial Fractions**

The natural frequencies of a system can be identified by expanding the system functional  $\mathcal{H}$  in partial fractions.

$$\mathcal{H} = \frac{Y}{X} = \frac{b_0 + b_1 \mathcal{R} + b_2 \mathcal{R}^2 + b_3 \mathcal{R}^3 + \cdots}{1 + a_1 \mathcal{R} + a_2 \mathcal{R}^2 + a_3 \mathcal{R}^3 + \cdots}$$

Factor denominator:

$$\mathcal{H} = \frac{Y}{X} = \frac{b_0 + b_1 \mathcal{R} + b_2 \mathcal{R}^2 + b_3 \mathcal{R}^3 + \cdots}{(1 - p_0 \mathcal{R})(1 - p_1 \mathcal{R})(1 - p_2 \mathcal{R})(1 - p_3 \mathcal{R}) \cdots}$$

Partial fractions:

$$\mathcal{H} = \frac{Y}{X} = \frac{C_0}{1 - p_0 \mathcal{R}} + \frac{C_1}{1 - p_1 \mathcal{R}} + \frac{C_2}{1 - p_2 \mathcal{R}} + \dots + D_0 + D_1 \mathcal{R} + D_2 \mathcal{R}^2 + \dots$$

One natural frequency  $(p_i^n)$  arises from each factor of the denominator. The polynomial terms  $(D_i)$  represent transient response components.

#### Poles

The form of each persistent mode is geometric, and the bases  $p_i$  of the geometrics are called the **poles** of the system.

$$\mathcal{H}(\mathcal{R}) = \frac{Y}{X} = \frac{C_0}{1 - p_0 \mathcal{R}} + \frac{C_1}{1 - p_1 \mathcal{R}} + \frac{C_2}{1 - p_2 \mathcal{R}} + \dots + D_0 + D_1 \mathcal{R} + D_2 \mathcal{R}^2 + \dots$$

Poles can be found by factoring the system functional  $\mathcal{H}(\mathcal{R})$  as shown above. But an easier way to find the poles is to solve for the roots of the denominator of the system function H(z):

$$H(z) = \mathcal{H}(\mathcal{R})\Big|_{\mathcal{R} \to \frac{1}{z}}$$

as shown below.

$$H(z) = \frac{C_0}{1 - p_0 z^{-1}} + \frac{C_1}{1 - p_1 z^{-1}} + \frac{C_2}{1 - p_2 z^{-1}} + \dots + D_0 + D_1 z^{-1} + D_2 z^{-2} + \dots$$
$$= \frac{C_0 z}{z - p_0} + \frac{C_1 z}{z - p_1} + \frac{C_2 z}{z - p_2} + \dots + D_0 + D_1 z^{-1} + D_2 z^{-2} + \dots$$

## **Check Yourself**

Consider the system described by

$$y[n] = -\frac{1}{4}y[n-1] + \frac{1}{8}y[n-2] + x[n-1] - \frac{1}{2}x[n-2]$$

How many of the following are true?

- 1. The unit sample response converges to zero.
- 2. There are poles at  $z = \frac{1}{2}$  and  $z = \frac{1}{4}$ .
- 3. There is a pole at  $z = \frac{1}{2}$ .
- 4. There are two poles.
- 5. None of the above

#### **Fibonacci's Bunnies**

Think about Fibonacci numbers as the output of a discrete-time system.

"How many pairs of rabbits can be produced from a single pair in a year if it is supposed that every month each pair begets a new pair which from the second month on becomes productive?"

Let c[n] represent the number of pairs of children in generation n. Assume that children become adults in one generation, so the total number of pairs of adults in generation n is the sum of the number of pairs of adults in generation n-1 plus the number of pairs of children in generation n-1.

$$a[n] = a[n-1] + c[n-1]$$

Each pair of adults produces a new pair of children in each generation, which adds to the number of pairs of children added externally (x[n]):

$$c[n]=x[n]+a[n{-}1]$$

Difference equation model:

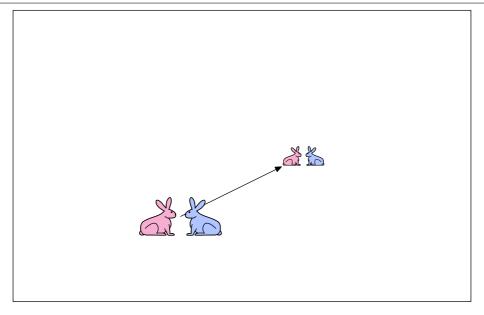
$$y[n] = a[n] + c[n] = y[n-1] + y[n-2] + x[n-1]$$

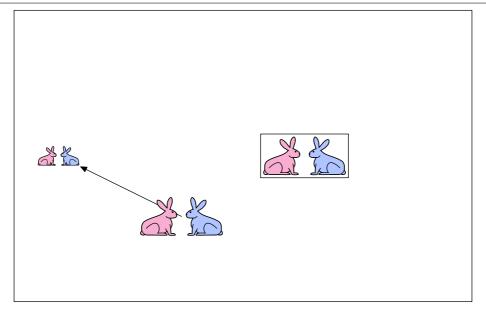
Start the population by adding one pair of children at n=0:

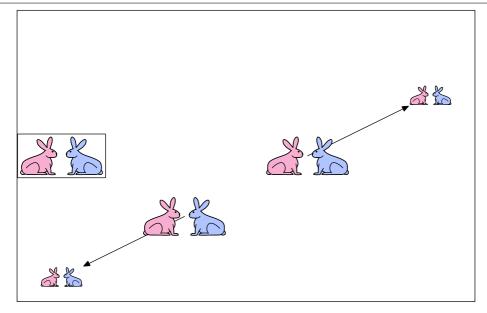
$$y[-1] = 0; \quad y[0] = 1$$

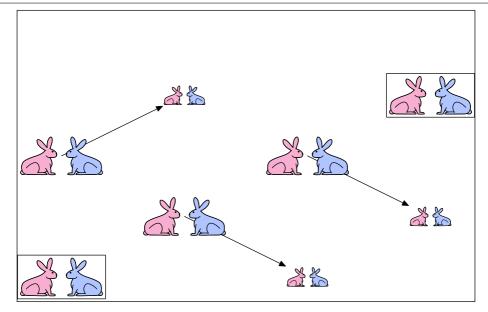


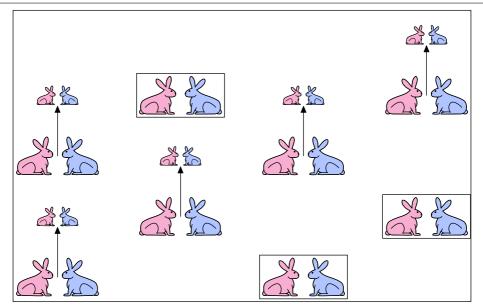


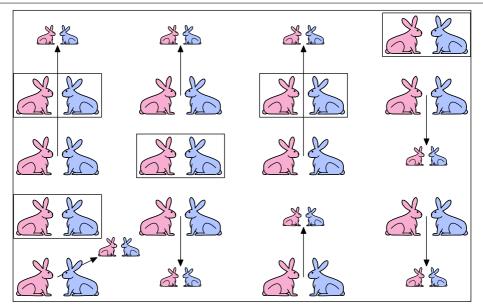


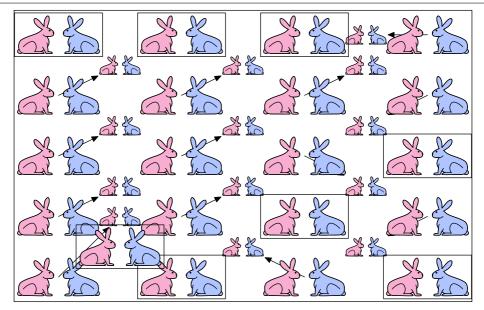


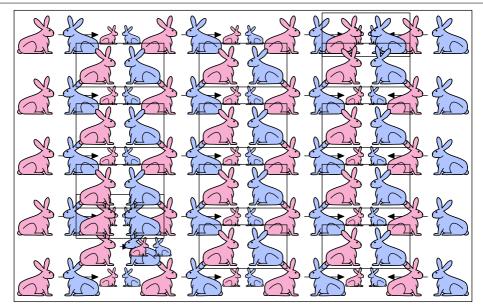












### **Check Yourself**

Bunnie system: y[n] = y[n-1] + y[n-2] + x[n-1]

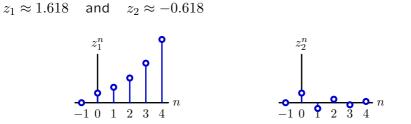
What are the pole(s) of the bunnie system?

**1**. 1

- 2. 1 and -1
- 3.  $-1 \ \mathrm{and} \ -2$
- 4. 1.618... and -0.618...
- 5. none of the above

#### Example: Fibonacci's Bunnies

Each pole corresponds to a natural frequency.



One mode diverges, one mode oscillates!

#### Summary

Today we characterized fundamental differences between feedforward and feedback systems.

- Feedforward systems can be characterized by a sum of components that are each characterized by an aggregate gain and delay.
- Feedback systems can be characterized by a ratio of polynomials in  $\mathcal{R}$  or equivalently by a ratio of polynomials in z.
- The natural frequencies of a feedback system are given by its **poles**, which are the roots of the denominator of the system function.