# 6.3100: Dynamic System Modeling and Control Design

# State-Space Responses

- Step Response
- Matrix Exponentials

November 04, 2024

#### State-Space Approach

Last week, we introduced the **State-Space** approach to control:

- Describe a system by its states.
- Describe dynamics of a system by **first-order** relations among states.
- Collect the states and relations in a single first-order **matrix** equation.

$$y_d(t) \rightarrow K_r \rightarrow + \underbrace{u(t)}_{\mathbf{k}(t)} \mathbf{\dot{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t) \xrightarrow{\mathbf{x}(t)} \mathbf{C} \rightarrow y(t)$$

Plant: state matrix  $\boldsymbol{\mathsf{A}}$ , input vector  $\boldsymbol{B}$ , and output vector  $\boldsymbol{C}$ :

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$$

$$y(t) = \mathbf{C}\mathbf{x}(t)$$

**Feedback** is characterized by a feedback vector **K** and input scaler  $K_r$ :

$$u(t) = K_r y_d(t) - \mathbf{K} \mathbf{x}(t)$$

Combine to obtain **closed-loop** characterization:

$$\dot{\mathbf{x}}(t) = \left(\mathbf{A} - \mathbf{B}\mathbf{K}\right)\mathbf{x}(t) + \mathbf{B}\mathbf{K}_{\mathbf{r}}y_d(t) \equiv \mathbf{A}_{\mathbf{c}}\mathbf{x}(t) + \mathbf{B}_{\mathbf{c}}y_d(t)$$

#### From State-Space to Transfer Function

Find the transfer function representation from the state-space description (similar to finding natural frequencies but  $\mathbf{B_c} \neq 0$ ).

Start with the state equation:

 $\dot{\mathbf{x}}(t) = \mathbf{A_c}\mathbf{x}(t) {+} \mathbf{B_c}u(t)$ 

Consider the input u(t) and state  $\mathbf{x}(t)$  at a particular complex frequency s:

$$u(t) = U(s)e^{st}$$
 and  $\mathbf{x}(t) = \mathbf{X}(s)e^{st}$ 

Find H(s) at the same complex frequency.

$$s\mathbf{X}(s)e^{st} = \mathbf{A_c}\mathbf{X}(s)e^{st} + \mathbf{B_c}U(s)e^{st}$$
$$s\mathbf{X}(s) = \mathbf{A_c}\mathbf{X}(s) + \mathbf{B_c}U(s)$$
$$(s\mathbf{I}-\mathbf{A_c})\mathbf{X}(s) = \mathbf{B_c}U(s)$$
$$\mathbf{X}(s) = (s\mathbf{I}-\mathbf{A_c})^{-1}\mathbf{B_c}U(s)$$
$$\mathbf{Y}(s) = \mathbf{C_c}\mathbf{X}(s) = \mathbf{C_c}(s\mathbf{I}-\mathbf{A_c})^{-1}\mathbf{B_c}U(s)$$
$$H(s) = \frac{Y(s)}{U(s)} = \mathbf{C_c}(s\mathbf{I}-\mathbf{A_c})^{-1}\mathbf{B_c}$$

#### From State-Space to Transfer Function

Example: find the open-loop transfer function

$$H(s) = \frac{Y(s)}{U(s)} = \mathbf{C}(s\mathbf{I}-\mathbf{A})^{-1}\mathbf{B}$$

for mass-spring-dashpot system.

$$\mathbf{A} = \begin{bmatrix} 0 & 1 \\ -k/m & -b/m \end{bmatrix} \quad \mathbf{B} = \begin{bmatrix} 0 \\ k/m \end{bmatrix} \quad \mathbf{C} = \begin{bmatrix} 1 & 0 \end{bmatrix}$$
$$H(s) = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} s & -1 \\ k/m & s+b/m \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ k/m \end{bmatrix}$$
$$= \begin{bmatrix} 1 & 0 \end{bmatrix} \frac{1}{s^2 + sb/m + k/m} \begin{bmatrix} s+b/m & 1 \\ -k/m & s \end{bmatrix} \begin{bmatrix} 0 \\ k/m \end{bmatrix}$$
$$= \frac{1}{s^2 + sb/m + k/m} \begin{bmatrix} s+b/m & 1 \end{bmatrix} \begin{bmatrix} 0 \\ k/m \end{bmatrix}$$
$$= \frac{k/m}{s^2 + sb/m + k/m} \quad \checkmark$$

The denominator (and therefore poles) come from  $\left| s \mathbf{I} - \mathbf{A} \right|$ .

#### State-Space Analysis of Natural Frequencies

Are there frequencies s for which large outputs result when input u(t)=0?

$$H(s) = \frac{Y(s)}{X(s)} = \mathbf{C_c}(s\mathbf{I} - \mathbf{A})^{-1}\mathbf{B_c} = \mathbf{C_c} \frac{\operatorname{adj}(s\mathbf{I} - \mathbf{A})}{|s\mathbf{I} - \mathbf{A}|} \mathbf{B_c}$$

If  $|s\mathbf{I}-\mathbf{A}|=0$ , H(s) is unbounded and therefore  $|Y(s)| \to \infty$ .

The natural frequencies are the solutions to the characteristic equation:  $|s \mathbf{I} - \mathbf{A_c}| = 0$ 

Example: mass-spring-dashpot system:

$$\mathbf{A_c} = \mathbf{A} - \mathbf{B}\mathbf{K} = \begin{bmatrix} 0 & 1 \\ -k/m & -b/m \end{bmatrix} - \begin{bmatrix} 0 \\ k/m \end{bmatrix} \begin{bmatrix} K_1 & K_2 \end{bmatrix}$$
$$|s \mathbf{I} - \mathbf{A_c}| = \left| \begin{bmatrix} s & -1 \\ k(1+K_1)/m & s+(b+kK_2)/m \end{bmatrix} \right| = 0$$

Characteristic equation:  $s^2 + (b+kK_2)s/m + k(1+K_1)/m = 0$   $\sqrt{}$ 

A state-space controller is represented by the following equations:

 $\begin{aligned} \dot{\mathbf{x}}(t) &= \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t) \\ u(t) &= K_r y_d(t) - \mathbf{K}\mathbf{x}(t) \\ y(t) &= \mathbf{C}\mathbf{x}(t) \end{aligned}$ 

Which block diagrams (below) correspond to the equations?



A state-space controller is represented by the following equations:

 $\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$  $u(t) = K_r y_d(t) - \mathbf{K}\mathbf{x}(t)$  $y(t) = \mathbf{C}\mathbf{x}(t)$ 

Which block diagrams (below) correspond to the equations? all



#### Step Response

Find the step response  $\mathbf{x}_{s}(t)$  of the following matrix system equation:

$$\dot{\mathbf{x}}(t) = \mathbf{P}\mathbf{x}(t) + \mathbf{Q}u(t)$$

Assume the initial value of the step response  $\mathbf{x}_{\mathbf{s}}(0) = \mathbf{0}$ , and u(t)=1 for t>0.

Homogeneous equation:  $\dot{\mathbf{x}}_{\mathbf{h}}(t) = \mathbf{P}\mathbf{x}_{\mathbf{h}}(t)$ 

#### Step Response

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Homogeneous equation:  $\dot{\mathbf{x}}_{\mathbf{h}}(t) = \mathbf{P}\mathbf{x}_{\mathbf{h}}(t)$ 

If this were a scalar equation:

 $\dot{x}_h(t) = p x_h(t)$ 

then the solution would be an exponential function of time:

$$x_h(t) = \alpha e^{pt}$$

Is there a matrix version of the exponential time function  $e^{pt}$ ?

#### **Scalar Exponential Function**

Exponential functions are eigenfunctions of the derivative operator:

$$e^{pt} \longrightarrow \frac{d}{dt} \longrightarrow pe^{pt}$$

Express the exponential function as a power series:

$$e^{pt} = 1 + \frac{pt}{1!} + \frac{p^2t^2}{2!} + \frac{p^3t^3}{3!} + \cdots$$

Differentiate term-by-term:

$$\begin{aligned} \frac{d}{dt}e^{pt} &= 0 + \frac{p}{1!} + \frac{2p^2t}{2!} + \frac{3p^3t^2}{3!} + \cdots \\ &= p + p\frac{pt}{1!} + p\frac{p^2t^2}{2!} + p\frac{p^3t^3}{3!} + \cdots \\ &= p\left(1 + \frac{pt}{1!} + \frac{p^2t^2}{2!} + \frac{p^3t^3}{3!} + \cdots\right) \\ &= pe^{pt} \quad \checkmark \end{aligned}$$

#### **Matrix Exponential Function**

Matrix exponentials are eigenfunctions of the matrix derivative operator:

$$e^{\mathbf{P}t} \longrightarrow \frac{d}{dt} \longrightarrow \mathbf{P}e^{\mathbf{P}t}$$

The matrix exponential function can also be expanded as a power series:  $e^{\mathbf{P}t} = \mathbf{I} + \frac{\mathbf{P}t}{1!} + \frac{\mathbf{P}^2t^2}{2!} + \frac{\mathbf{P}^3t^3}{3!} + \cdots$ 

Differentiate term-by-term:

1

$$\begin{aligned} \frac{d}{dt}e^{\mathbf{P}t} &= 0 + \frac{\mathbf{P}}{1!} + \frac{2\mathbf{P}^{2}t}{2!} + \frac{3\mathbf{P}^{3}t^{2}}{3!} + \cdots \\ &= \mathbf{P} + \mathbf{P}\frac{\mathbf{P}t}{1!} + \mathbf{P}\frac{\mathbf{P}^{2}t^{2}}{2!} + \mathbf{P}\frac{\mathbf{P}^{3}t^{3}}{3!} + \cdots \\ &= \mathbf{P}\left(\mathbf{I} + \frac{\mathbf{P}t}{1!} + \frac{\mathbf{P}^{2}t^{2}}{2!} + \frac{\mathbf{P}^{3}t^{3}}{3!} + \cdots\right) \\ &= \mathbf{P}e^{\mathbf{P}t} = e^{\mathbf{P}t}\mathbf{P} \quad \checkmark \end{aligned}$$

#### Step Response

Find the step response  $\mathbf{x}_{s}(t)$  of the following matrix system equation:

$$\dot{\mathbf{x}}(t) = \mathbf{P}\mathbf{x}(t) + \mathbf{Q}u(t)$$

Assume the initial value of the step response  $\mathbf{x}_{\mathbf{s}}(0) = \mathbf{0}$ , and u(t)=1 for t>0.

Homogeneous equation:  $\dot{\mathbf{x}}_{\mathbf{h}}(t) = \mathbf{P}\mathbf{x}_{\mathbf{h}}(t)$ 

$$\mathbf{x}_{\mathbf{h}}(t) = e^{\mathbf{P}t} \mathbf{\Psi}$$

Particular solution:  $\mathbf{x}_{\mathbf{p}}(t) = \mathbf{\Phi}$ 

$$\dot{\mathbf{x}}_{\mathbf{p}}(t) = \mathbf{0} = \mathbf{P} \mathbf{\Phi} + \mathbf{Q}$$
  
 $\mathbf{\Phi} = -\mathbf{P}^{-1}\mathbf{Q}$  (provided that  $\mathbf{P}$  is not singular)

Initial condition:  $\mathbf{x}(0) = \mathbf{\Psi} - \mathbf{P^{-1}Q} = \mathbf{0}$ 

$$\Psi = P^{-1} Q$$

Step response:

$$\begin{aligned} \mathbf{x}_{\mathbf{s}}(t) &= (e^{\mathbf{P}t} - \mathbf{I})\mathbf{P}^{-1}\mathbf{Q} \\ &= \mathbf{P}^{-1}(e^{\mathbf{P}t} - \mathbf{I})\mathbf{Q} \end{aligned}$$

Exponential functions play important role in solving matrix diff eq's.

Finding the series expansion of a matrix exponential

$$e^{\mathbf{P}} = \mathbf{I} + \mathbf{P} + \mathbf{P}^2/2! + \mathbf{P}^3/3! + \mathbf{P}^4/4! + \cdots$$

is easy when P is diagonal:



Fortunately it's easy to **diagonalize** a matrix that is full-rank and has distinct eigenvalues. Start with the eigenvector/eigenvalue property:

$$\mathbf{P}\mathbf{v}_{\mathbf{i}} = \lambda_i \mathbf{v}_{\mathbf{i}}$$

where  $\lambda_i$  is the  $i^{\text{th}}$  eigenvalue and  $\mathbf{v_i}$  is the  $i^{\text{th}}$  eigenvector (a column vector). If **P** is full rank and if none of the eigenvalues are repeated

$$\begin{split} \mathbf{P} \begin{bmatrix} \mathbf{v_1} | \mathbf{v_2} | \mathbf{v_3} | \cdots | \mathbf{v_n} \end{bmatrix} &= \begin{bmatrix} \mathbf{P} \mathbf{v_1} | \mathbf{P} \mathbf{v_2} | \mathbf{P} \mathbf{v_3} | \cdots | \mathbf{P} \mathbf{v_n} \end{bmatrix} \\ &= \begin{bmatrix} \lambda_1 \mathbf{v_1} | \lambda_2 \mathbf{v_2} | \lambda_3 \mathbf{v_3} | \cdots | \lambda_n \mathbf{v_n} \end{bmatrix} \\ &= \begin{bmatrix} \mathbf{v_1} | \mathbf{v_2} | \mathbf{v_3} | \cdots | \mathbf{v_n} \end{bmatrix} \begin{bmatrix} \lambda_1 & & \\ & \lambda_2 & \\ & & \ddots \end{bmatrix} \\ & \mathbf{P} \mathbf{V} = \mathbf{V} \mathbf{\Lambda} \\ & \mathbf{P} = \mathbf{V} \mathbf{\Lambda} \mathbf{V}^{-1} \\ & & & & \mathbf{N} = \begin{bmatrix} \mathbf{v_1} | \mathbf{v_2} | \mathbf{v_3} | \cdots | \mathbf{v_n} \end{bmatrix} \text{ and } \mathbf{\Lambda} = \begin{bmatrix} \lambda_1 & & \\ & \lambda_2 & \\ & & \ddots \end{bmatrix} \end{split}$$

Substitute the diagonal expansion of  $P\colon$ 

$$\mathbf{P} = \mathbf{V} \mathbf{\Lambda} \mathbf{V}^{-1}$$

$$e^{\mathbf{P}} = \mathbf{I} + \mathbf{P} + \mathbf{P}^{2}/2! + \mathbf{P}^{3}/3! + \mathbf{P}^{4}/4! + \cdots$$

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$$e^{\mathbf{P}} = \mathbf{I} + \mathbf{P} + \mathbf{P}^{2}/2! + \mathbf{P}^{3}/3! + \mathbf{P}^{4}/4! + \cdots$$
$$= \mathbf{I} + \mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1}\mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1}\mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1} + \cdots$$

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=  $\mathbf{I} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \cdots$   
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=  $\mathbf{I} + \mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1}\mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1}\mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1} + \cdots$   
=  $\mathbf{I} + \mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\mathbf{\Lambda}^{2}\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\mathbf{\Lambda}^{3}\mathbf{V}^{-1} + \cdots$   
=  $\mathbf{V}\mathbf{V}^{-1} + \mathbf{V}\mathbf{\Lambda}\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\mathbf{\Lambda}^{2}\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\mathbf{\Lambda}^{3}\mathbf{V}^{-1} + \cdots$ 

Substitute the diagonal expansion of  $P\colon$ 

$$\mathbf{P} = \mathbf{V} \mathbf{\Lambda} \mathbf{V}^{-1}$$

$$\begin{split} e^{\mathbf{P}} &= \mathbf{I} + \mathbf{P} + \mathbf{P}^{2}/2! + \mathbf{P}^{3}/3! + \mathbf{P}^{4}/4! + \cdots \\ &= \mathbf{I} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \cdots \\ &= \mathbf{I} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \cdots \\ &= \mathbf{I} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda^{2}\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda^{3}\mathbf{V}^{-1} + \cdots \\ &= \mathbf{V}\mathbf{V}^{-1} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda^{2}\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda^{3}\mathbf{V}^{-1} + \cdots \\ &= \mathbf{V}\left(\mathbf{I} + \Lambda + \frac{1}{2!}\Lambda^{2} + \frac{1}{3!}\Lambda^{3} + \cdots\right)\mathbf{V}^{-1} \end{split}$$

Substitute the diagonal expansion of  $P\colon$ 

$$\mathbf{P} = \mathbf{V} \mathbf{\Lambda} \mathbf{V}^{-1}$$

into the series expansion of  $e^{\mathbf{P}}$ :

$$\begin{split} e^{\mathbf{P}} &= \mathbf{I} + \mathbf{P} + \mathbf{P}^{2}/2! + \mathbf{P}^{3}/3! + \mathbf{P}^{4}/4! + \cdots \\ &= \mathbf{I} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \cdots \\ &= \mathbf{I} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1}\mathbf{V}\Lambda\mathbf{V}^{-1} + \cdots \\ &= \mathbf{I} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda^{2}\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda^{3}\mathbf{V}^{-1} + \cdots \\ &= \mathbf{V}\mathbf{V}^{-1} + \mathbf{V}\Lambda\mathbf{V}^{-1} + \frac{1}{2!}\mathbf{V}\Lambda^{2}\mathbf{V}^{-1} + \frac{1}{3!}\mathbf{V}\Lambda^{3}\mathbf{V}^{-1} + \cdots \\ &= \mathbf{V}\left(\mathbf{I} + \Lambda + \frac{1}{2!}\Lambda^{2} + \frac{1}{3!}\Lambda^{3} + \cdots\right)\mathbf{V}^{-1} \\ &= \mathbf{V}e^{\mathbf{\Lambda}}\mathbf{V}^{-1} \end{split}$$

The matrix exponential of  ${\bf P}$  can be directly computed from the eigenvalues and eigenvectors of  ${\bf P}.$ 

Determine 
$$e^{\mathbf{P}t}$$
 for  $\mathbf{P} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$ .

Step 1: Find the eigenvalues of P.

Determine 
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Step 1: Find the eigenvalues of P.

$$|s\mathbf{I} - \mathbf{P}| = \left| \begin{bmatrix} s & -1\\ 2 & s+3 \end{bmatrix} \right| = s^2 + 3s + 2 = (s+1)(s+2) = 0$$
$$s_{1,2} = -1, -2$$

Determine 
$$e^{\mathbf{P}t}$$
 for  $\mathbf{P} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$ .

Step 1: Find the eigenvalues of  $\mathbf{P}$ : -1 and -2.

$$\mathbf{P}\mathbf{v} = \mathbf{P}\begin{bmatrix} a\\b\end{bmatrix} = s\mathbf{v} = s\begin{bmatrix} a\\b\end{bmatrix}$$

Determine 
$$e^{\mathbf{P}t}$$
 for  $\mathbf{P} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$ .

Step 1: Find the eigenvalues of  $\mathbf{P}$ : -1 and -2.

$$\mathbf{Pv} = \mathbf{P} \begin{bmatrix} a \\ b \end{bmatrix} = s\mathbf{v} = s \begin{bmatrix} a \\ b \end{bmatrix}$$
$$s = -1:$$
$$\begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} \begin{bmatrix} a \\ b \end{bmatrix} = -1 \begin{bmatrix} a \\ b \end{bmatrix}$$
$$b = -a$$
$$-2a - 3b = -b$$
$$\begin{bmatrix} a \\ b \end{bmatrix} = \alpha \begin{bmatrix} 1 \\ -1 \end{bmatrix}; \ \forall \alpha$$

Determine 
$$e^{\mathbf{P}t}$$
 for  $\mathbf{P} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$ .

Step 1: Find the eigenvalues of  $\mathbf{P}$ : -1 and -2.

$$\mathbf{Pv} = \mathbf{P} \begin{bmatrix} a \\ b \end{bmatrix} = s\mathbf{v} = s \begin{bmatrix} a \\ b \end{bmatrix}$$
$$s = -2:$$
$$\begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} \begin{bmatrix} a \\ b \end{bmatrix} = -2 \begin{bmatrix} a \\ b \end{bmatrix}$$
$$b = -2a$$
$$-2a - 3b = -2b$$
$$\begin{bmatrix} a \\ b \end{bmatrix} = \beta \begin{bmatrix} 1 \\ -2 \end{bmatrix}; \ \forall \beta$$

Determine 
$$e^{\mathbf{P}t}$$
 for  $\mathbf{P} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$ .

Step 1: Find the eigenvalues of  $\mathbf{P}$ : -1 and -2.

Step 2: Find the eigenvectors of **P**: 
$$\begin{bmatrix} 1 \\ -1 \end{bmatrix}$$
 and  $\begin{bmatrix} 1 \\ -2 \end{bmatrix}$ .

Determine 
$$e^{\mathbf{P}t}$$
 for  $\mathbf{P} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$ .

Step 1: Find the eigenvalues of P: -1 and -2.

Step 2: Find the eigenvectors of  $\mathbf{P}$ :  $\begin{bmatrix} 1 \\ -1 \end{bmatrix}$  and  $\begin{bmatrix} 1 \\ -2 \end{bmatrix}$ .

Step 3: Find  $e^{\mathbf{P}t} = \mathbf{V}e^{\mathbf{\Lambda}t}\mathbf{V}^{-1}$ .

$$e^{\mathbf{P}t} = \begin{bmatrix} 1 \\ -1 \\ -1 \end{bmatrix} \begin{bmatrix} e^{-t} & 0 \\ 0 & e^{-2t} \end{bmatrix} \begin{bmatrix} 1 \\ -1 \\ -1 \end{bmatrix} \begin{bmatrix} -1 \\ -2 \end{bmatrix}^{-1}$$
$$= \begin{bmatrix} 1 & 1 \\ -1 & -2 \end{bmatrix} \begin{bmatrix} e^{-t} & 0 \\ 0 & e^{-2t} \end{bmatrix} \begin{bmatrix} 2 & 1 \\ -1 & -1 \end{bmatrix}$$
$$= \begin{bmatrix} e^{-t} & e^{-2t} \\ -e^{-t} & -2e^{-2t} \end{bmatrix} \begin{bmatrix} 2 & 1 \\ -1 & -1 \end{bmatrix}$$
$$= \begin{bmatrix} 2e^{-t} - e^{-2t} & e^{-t} - e^{-2t} \\ -2e^{-t} + 2e^{-2t} & -e^{-t} + 2e^{-2t} \end{bmatrix}$$

Let  $\mathbf{P} = \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix}$ 

Which of the following matrices equals  $e^{\mathbf{P}t}$ ?

1. 
$$\begin{bmatrix} e^{jt} & e^{-jt} \\ e^{-jt} & e^{jt} \end{bmatrix}$$
2. 
$$\begin{bmatrix} \cos(t) & \sin(t) \\ -\sin(t) & \cos(t) \end{bmatrix}$$
3. 
$$\begin{bmatrix} e^t & e^{-t} \\ e^{-t} & e^t \end{bmatrix}$$
4. 
$$\begin{bmatrix} te^t & te^{-t} \\ te^{-t} & te^t \end{bmatrix}$$

5. none of the above

Let

$$\mathbf{P} = \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix}$$

Determine  $e^{\mathbf{P}t}$ 

Step 1: Find the eigenvalues of  $\boldsymbol{P}.$ 

$$|s\mathbf{I} - \mathbf{P}| = 0$$
$$\left| \begin{bmatrix} s & 0\\ 0 & s \end{bmatrix} - \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix} \right| = \left| \begin{bmatrix} s & -1\\ 1 & s \end{bmatrix} \right| = s^2 + 1 = 0$$
$$s_{1,2} = \pm j$$

Let

$$\mathbf{P} = \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix}$$

Determine  $e^{\mathbf{P}t}$ 

Step 1: Find the eigenvalues of  ${\bf P}:~s_{1,2}=\pm j$ 

.

Let

$$\mathbf{P} = \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix}$$

Determine  $e^{\mathbf{P}t}$ 

Step 1: Find the eigenvalues of  ${\bf P}:~s_{1,2}=\pm j$ 

Step 2: Find the eigenvectors of  $\boldsymbol{P}$ 

$$\mathbf{Pv} = s\mathbf{v}$$

$$\begin{bmatrix} 0 & 1 \\ -1 & 0 \end{bmatrix} \begin{bmatrix} a \\ b \end{bmatrix} = j \begin{bmatrix} a \\ b \end{bmatrix}$$

$$b = ja$$

$$-a = jb$$

The second equation is j times the first. The eigenvector can be taken as any scalar multiple of  $\begin{bmatrix} 1\\ j \end{bmatrix}$ 

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$$\mathbf{Pv} = s\mathbf{v}$$

$$\begin{bmatrix} 0 & 1 \\ -1 & 0 \end{bmatrix} \begin{bmatrix} a \\ b \end{bmatrix} = -j \begin{bmatrix} a \\ b \end{bmatrix}$$

$$b = -ja$$

$$-a = -jb$$

The second equation is -j times the first. The eigenvector can be taken as any scalar multiple of  $\begin{bmatrix}1\\-j\end{bmatrix}$ 

Let

 $\mathbf{P} = \begin{bmatrix} 0 & 1 \\ -1 & 0 \end{bmatrix}$ 

Determine  $e^{\mathbf{P}t}$ 

Step 1: Find the eigenvalues of  $\mathbf{P} \text{:}~ s_{1,2} = \pm j$ 

Step 2: Find the eigenvectors of 
$$\mathbf{P}$$
:  $\begin{bmatrix} 1\\ j \end{bmatrix}$  and  $\begin{bmatrix} 1\\ -j \end{bmatrix}$ .

Step 3: Calculate the complex exponential.

Let

$$\mathbf{P} = \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix}$$

Determine  $e^{\mathbf{P}t}$ 

Step 1: Find the eigenvalues of  $\mathbf{P}:~s_{1,2}=\pm j$ 

Step 2: Find the eigenvectors of 
$$\mathbf{P}$$
:  $\begin{bmatrix} 1 \\ j \end{bmatrix}$  and  $\begin{bmatrix} 1 \\ -j \end{bmatrix}$ .

Step 3: Calculate the complex exponential.

$$e^{\mathbf{P}t} = \mathbf{V}e^{\mathbf{\Lambda}t}\mathbf{V}^{-1} = \begin{bmatrix} 1 & 1\\ j & -j \end{bmatrix} \begin{bmatrix} e^{jt} & 0\\ 0 & e^{-jt} \end{bmatrix} \begin{bmatrix} 1/2 & -j/2\\ 1/2 & j/2 \end{bmatrix}$$
$$= \begin{bmatrix} e^{jt} & e^{-jt}\\ je^{jt} & -je^{-jt} \end{bmatrix} \begin{bmatrix} 1/2 & -j/2\\ 1/2 & j/2 \end{bmatrix} = \begin{bmatrix} (e^{jt}+e^{-jt})/2 & -j(e^{jt}-e^{-jt})/2\\ j(e^{jt}-e^{-jt})/2 & (e^{jt}+e^{-jt})/2 \end{bmatrix}$$
$$= \begin{bmatrix} \cos(t) & \sin(t)\\ -\sin(t) & \cos(t) \end{bmatrix}$$

Let  $\mathbf{P} = \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix}$ 

Which of the following matrices equals  $e^{\mathbf{P}t}$ ? 2

1. 
$$\begin{bmatrix} e^{jt} & e^{-jt} \\ e^{-jt} & e^{jt} \end{bmatrix}$$
  
2.  $\begin{bmatrix} \cos(t) & \sin(t) \\ -\sin(t) & \cos(t) \end{bmatrix}$   
3.  $\begin{bmatrix} e^t & e^{-t} \\ e^{-t} & e^t \end{bmatrix}$   
4.  $\begin{bmatrix} te^t & te^{-t} \\ te^{-t} & te^t \end{bmatrix}$ 

5. none of the above

Let

$$\mathbf{P} = \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix}$$

Determine  $e^{\mathbf{P}t}$ 

Alternative method: use series expansion

$$\begin{split} e^{\mathbf{P}t} &= \mathbf{I} + \frac{\mathbf{P}t}{1!} + \frac{\mathbf{P}^{2}t^{2}}{2!} + \frac{\mathbf{P}^{3}t^{3}}{3!} + \frac{\mathbf{P}^{4}t^{4}}{4!} + \cdots \\ &= \begin{bmatrix} 1 & 0\\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix} t + \begin{bmatrix} -1 & 0\\ 0 & -1 \end{bmatrix} \frac{t^{2}}{2!} + \begin{bmatrix} 0 & -1\\ 1 & 0 \end{bmatrix} \frac{t^{3}}{3!} + \begin{bmatrix} 1 & 0\\ 0 & 1 \end{bmatrix} \frac{t^{4}}{4!} + \cdots \\ &= \begin{bmatrix} 1 - \frac{t^{2}}{2!} + \frac{t^{4}}{4!} - \frac{t^{6}}{6!} + \frac{t^{8}}{8!} - + \cdots & t - \frac{t^{3}}{3!} + \frac{t^{5}}{5!} - \frac{t^{7}}{7!} + \frac{t^{9}}{9!} - + \cdots \\ &- t + \frac{t^{3}}{3!} - \frac{t^{5}}{5!} + \frac{t^{7}}{7!} - \frac{t^{9}}{9!} \cdots & 1 - \frac{t^{2}}{2!} + \frac{t^{4}}{4!} - \frac{t^{6}}{6!} + \frac{t^{8}}{8!} - + \cdots \end{bmatrix} \\ &= \begin{bmatrix} \cos(t) & \sin(t) \\ -\sin(t) & \cos(t) \end{bmatrix} \end{split}$$

## **Compare Methods**

Let

$$\mathbf{P} = \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix}$$

Determine  $e^{\mathbf{P}t}$ 

Series method requires closing infinite sums.  $\xspace{\xspace{X}}$ 

$$e^{\mathbf{P}t} = \mathbf{I} + \frac{\mathbf{P}t}{1!} + \frac{\mathbf{P}^{2}t^{2}}{2!} + \frac{\mathbf{P}^{3}t^{3}}{3!} + \frac{\mathbf{P}^{4}t^{4}}{4!} + \cdots$$

$$= \begin{bmatrix} 1 & 0\\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & 1\\ -1 & 0 \end{bmatrix} t + \begin{bmatrix} -1 & 0\\ 0 & -1 \end{bmatrix} \frac{t^{2}}{2!} + \begin{bmatrix} 0 & -1\\ 1 & 0 \end{bmatrix} \frac{t^{3}}{3!} + \begin{bmatrix} 1 & 0\\ 0 & 1 \end{bmatrix} \frac{t^{4}}{4!} + \cdots$$

$$= \begin{bmatrix} 1 - \frac{t^{2}}{2!} + \frac{t^{4}}{4!} - \frac{t^{6}}{6!} + \frac{t^{8}}{8!} - + \cdots & t - \frac{t^{3}}{3!} + \frac{t^{5}}{5!} - \frac{t^{7}}{7!} + \frac{t^{9}}{9!} - + \cdots \\ -t + \frac{t^{3}}{3!} - \frac{t^{5}}{5!} + \frac{t^{7}}{7!} - \frac{t^{9}}{9!} \cdots & 1 - \frac{t^{2}}{2!} + \frac{t^{4}}{4!} - \frac{t^{6}}{6!} + \frac{t^{8}}{8!} - + \cdots \end{bmatrix}$$

$$= \begin{bmatrix} \cos(t) & \sin(t) \\ -\sin(t) & \cos(t) \end{bmatrix}$$

Let

 $\mathbf{P} = \begin{bmatrix} 0 & 1 \\ -1 & 0 \end{bmatrix}$ 

Determine  $e^{\mathbf{P}t}$ 

The diagonalizing method uses conventional linear algebra.  $\sqrt{}$ 

Step 1: Find the eigenvalues  $\mathbf{P}:~s_{1,2}=\pm j$ 

Step 2: Find the eigenvectors of  $\mathbf{P}$ :  $\begin{bmatrix} 1\\ j \end{bmatrix}$  and  $\begin{bmatrix} 1\\ -j \end{bmatrix}$ .

Step 3: Calculate the complex exponential  $e^{\mathbf{P}t} = \mathbf{V}e^{\mathbf{\Lambda}t}\mathbf{V}^{-1}$ 

- assemble  $\mathbf{V} =$  matrix of eigenvectors
- assemble  $e^{\mathbf{\Lambda}t} =$  matrix of eigenfunctions
- determine the inverse of V
- matrix multiply

Easily done with numerical methods in Python, MATLAB, ...

# **Controller Design**

Optimizing the gains  $\mathbf{K}$  and  $K_r$  of a state-space controller.

$$y_d(t) \rightarrow K_r \rightarrow + \underbrace{u(t)}_{\mathbf{x}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)} \underbrace{\mathbf{x}(t)}_{\mathbf{K}} \mathbf{x}(t) \rightarrow \mathbf{C} \rightarrow y(t)$$

#### Controlling a Mass-Spring-Dashpot

Start by reviewing how we chose gains for a classical controller.

Model the mass-spring-dashpot system (Newton's Law):

$$\underbrace{\frac{k\Big(x(t)-y(t)\Big)-b\dot{y}(t)}_{F}}_{F} = \underbrace{m\ddot{y}(t)}_{ma}$$

to get the (open loop) transfer function of the plant:

$$G(s) = \frac{Y(s)}{X(s)} = \frac{k}{s^2m + sb + k}$$

## Proportional Control of a Mass-Spring-Dashpot

Start by reviewing how we chose gains for a classical controller.



Closed-loop transfer function:

$$H(s) = \frac{Y(s)}{Y_d(s)} = \frac{kK_p}{s^2m + sb + k(1+K_p)}$$

Closed-loop poles are roots of denominator:  $s_1, s_2 =$ 

$$\frac{-b \pm \sqrt{b^2 - 4mk(1+K_p)}}{2m}$$

#### **Root-Locus Analysis of Proportional Control**



As  $K_p \uparrow$ , frequency of ringing  $\uparrow$ , but peaks decay with same time constant.

## **Proportional Plus Derivative Control**

Adding a derivative term helps.



Same open-loop transfer function, different controller:

$$Y_d(s) \longrightarrow (+) \longrightarrow K_p + sK_d \xrightarrow{X(s)} \xrightarrow{k} Y(s)$$

Closed-loop transfer function:

$$H(s) = \frac{Y(s)}{Y_d(s)} = \frac{k(K_p + sK_d)}{s^2m + s(b + kK_d) + k(1 + K_p)}$$

Closed-loop poles:  $s = \frac{-b - kK_d \pm \sqrt{(b + kK_d)^2 - 4mk(1 + K_p)}}{2m}$ 

#### **Root-Locus Analysis of PD Control**

Increasing  $K_d$  enables faster closed-loop poles (red dots) without overshoot.



### More Advanced Methods in Classical Control

Derivative feedback is just one way to optimize a classical controller.

Other advanced classical methods include

- integral feedback for PID control,
- optimizing gain and phase margins,
- lead compensation, lag compensation, lead/lag compensation, and
- many other techniques.

Much of the design power of these advanced methods results from their ability to **move poles and zeros** to locations that are more favorable for

- stability,
- disturbance rejection,
- noise immunity, etc.

We can similarly optimize state-space controllers.

And the state-space formulation is much more powerful!

# **Stopped Here**

### Check Yourself: Proportional Control in State Space



### Check Yourself: Proportional Control in State Space

Choose **K** and  $K_r$  so that the state-space controller:



is equivalent to a proportional controller:

$$y_d(t) \longrightarrow H$$
  $K_p$   $u(t)$   $G(s) = \frac{Y(s)}{U(s)}$   $y(t)$ 

To be equivalent, the controllers must produce the same plant input u(t):

$$u(t) = K_r y_d(t) - \mathbf{K} \mathbf{x}(t) = K_p (y_d(t) - y(t))$$
$$= K_p y_d(t) - K_p \mathbf{C} \mathbf{x}(t)$$

 $K_r = K_p$  $\mathbf{K} = K_p \mathbf{C}$ 

### Check Yourself: Proportional Control in State Space



### **Proportional Control with State-Space Model**

We can implement proportional control with a state-space model

$$y_d(t) \rightarrow K_r \rightarrow + u(t) \qquad \dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t) \qquad \mathbf{x}(t) \rightarrow \mathbf{C} \rightarrow y(t)$$

by setting  $\mathbf{K} = K_p \mathbf{C}$  and  $K_r = K_p$ :



We can also implement **proportional plus derivative control** with a statespace model.

### Proportional Plus Derivative Control in State-Space

Start with a classical system with PD control:

$$y_d(t) \longrightarrow + K_p + K_d \frac{d}{dt} \longrightarrow (classical description) \longrightarrow y(t)$$

Replace classical description of plant with equiv. state-space description:

$$y_d(t) \longrightarrow H_p + K_d \frac{d}{dt}$$
  $u(t)$  plant  $\mathbf{x}(t)$   $\mathbf{C} \longrightarrow y(t)$  (state-space desc)

Distribute the PD controller over the inputs to the subtractor:

$$y_{d}(t) \longrightarrow K_{p} + K_{d} \frac{d}{dt} \longrightarrow \underbrace{u(t)}_{(\text{state-space desc})} x(t) \xrightarrow{\mathbf{x}(t)}_{\mathbf{C}} y(t)$$
Feedback  $\mathbf{x}(t)$  instead of  $y(t)$ :
$$y_{d}(t) \longrightarrow K_{p} + K_{d} \frac{d}{dt} \longrightarrow \underbrace{u(t)}_{(\text{state-space desc})} x(t) \xrightarrow{\mathbf{x}(t)}_{\mathbf{C}} \mathbf{C} \longrightarrow y(t)$$

$$K_{p} - \underbrace{K_{p} - \underbrace{u(t)}_{(\text{state-space desc})} x(t)}_{K_{p} - \underbrace{K_{p} - \underbrace{K_{d} - \underbrace{d}_{dt}}_{dt}}$$

Result is a state-space controller with  $K_r = K_p + K_d \frac{d}{dt}$  and  $\mathbf{K} = K_p \mathbf{C} + K_d \mathbf{C} \frac{d}{dt}$ .

## More Advanced Methods in Classical Control

Much of the design power of the more advanced methods results from their ability to **move poles and zeros** to locations that are more favorable for

- stability,
- disturbance rejection,
- noise immunity, etc.

We can similarly optimize state-space controllers.

And the state-space formulation is much more powerful!

## **Pole Placement**

With the correct choice of gains **K** and  $K_r$ , we can move the closed-loop poles of a state-space model **anywhere** in the complex plane.



The closed-loop poles of a state-space model are equal to the roots of its characteristic polynomial:

$$|s\mathbf{I}-(\mathbf{A}-\mathbf{B}\mathbf{K})|=\mathbf{0}$$

**Fundamental theorem of algebra:** an  $n^{th}$  order polynomial as n roots. **Factor theorem:** each root determines a first-order factor.

 $\rightarrow$  characteristic polynomial can be written as a product of first-order terms:  $\left|s\mathbf{I}-(\mathbf{A}-\mathbf{B}\mathbf{K})\right| = \prod_{i=1}^{n}(s-s_i) = 0$ 

LHS:  $n^{th}$  order polynomial in s (pole locations)

RHS: same polynomial, but coeff's in terms of desired pole locations  $s_i$ .

Example: Pole Placement for Mass-Spring-Dashpot



$$\underbrace{k\Big(u(t)-y(t)\Big)-b\dot{y}(t)}_{F}=\underbrace{m\ddot{y}(t)}_{ma}$$

Rewrite this second-order differential equation as two **first-order** equations:

$$\underbrace{\frac{d}{dt} \begin{bmatrix} y(t) \\ \dot{y}(t) \end{bmatrix}}_{\dot{\mathbf{x}}(t)} = \underbrace{\begin{bmatrix} 0 & 1 \\ -k/m & -b/m \end{bmatrix}}_{\mathbf{A}} \underbrace{\begin{bmatrix} y(t) \\ \dot{y}(t) \end{bmatrix}}_{\mathbf{x}(t)} + \underbrace{\begin{bmatrix} 0 \\ k/m \end{bmatrix}}_{\mathbf{B}} u(t)$$

which can be expressed as a single matrix equation:

$$\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}u(t)$$

#### Example: Pole Placement for Mass-Spring-Dashpot

For 
$$m = 1$$
,  $b = 1.4$ , and  $k = 2$ :  

$$\mathbf{A} = \begin{bmatrix} 0 & 1 \\ -2 & -1.4 \end{bmatrix} \qquad \mathbf{B} = \begin{bmatrix} 0 \\ 2 \end{bmatrix}$$

With 
$$\mathbf{K} = \begin{bmatrix} k_1 & k_2 \end{bmatrix}$$
:  
 $|s\mathbf{I} - (\mathbf{A} - \mathbf{B}\mathbf{K})| = \left| \begin{bmatrix} s & -1 \\ 2+2k_1 & s+1.4+2k_2 \end{bmatrix} \right| = s^2 + (1.4+2k_2)s + (2+2k_1)$ 

To place poles at s = -0.5 and s = -1, set

$$s^{2}+(1.4+2k_{2})s+(2+2k_{1}) = (s+0.5)(s+1) = s^{2}+1.5s+0.5$$
  
 $\rightarrow k_{1} = -0.75$  and  $k_{2} = 0.05$ .

Alternatively, to place poles at  $\mathbf{s} = -0.5$  and  $\mathbf{s} = -0.6$ , set  $s^2 + (1.4+2k_2)s + (2+2k_1) = (s+0.5)(s+0.6) = s^2 + 1.1s + 0.3$  $\rightarrow k_1 = -0.85$  and  $k_2 = -0.15$ .

Let  ${\bf A}$  represent the system matrix and  ${\bf B}$  represent the input matrix for a state-space control system, where these matrices are given by

$$\mathbf{A} = \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix} \qquad \mathbf{B} = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

Which  $\mathbf{K}$  vector will produce closed-loop poles at 0 and -2?

1. 
$$\mathbf{K} = \begin{bmatrix} 1 & 2 \end{bmatrix}$$
 2.  $\mathbf{K} = \begin{bmatrix} 3 & 4 \end{bmatrix}$  3.  $\mathbf{K} = \begin{bmatrix} 1 \\ 2 \end{bmatrix}$  4.  $\mathbf{K} = \begin{bmatrix} 3 \\ 4 \end{bmatrix}$   
5. none of the above

$$\mathbf{A} = \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix} \qquad \mathbf{B} = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

Find  ${\bf K}$  so that the closed-loop poles are at 0 and -2.

$$|s\mathbf{I} - (\mathbf{A} - \mathbf{B}\mathbf{K})| = \left| \begin{bmatrix} s-1 & -1 \\ k_1 & s+k_2-1 \end{bmatrix} \right| = (s-1)(s+k_2-1)+k_1$$
$$= s^2 + (k_2-2)s + k_1 - k_2 + 1)$$
$$= s(s+2) = s^2 + 2s$$
$$k_2 - 2 = 2$$

$$k_1 - k_2 + 1 = 0$$

Solving, we find  $k_2 = 4$  and  $k_1 = 3$ .

Let  ${\bf A}$  represent the system matrix and  ${\bf B}$  represent the input matrix for a state-space control system, where these matrices are given by

$$\mathbf{A} = \begin{bmatrix} 1 & 1 \\ 0 & 1 \end{bmatrix} \qquad \mathbf{B} = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

Which K vector will produce closed-loop poles at 0 and -2? 2.

1. 
$$\mathbf{K} = \begin{bmatrix} 1 & 2 \end{bmatrix}$$
 2.  $\mathbf{K} = \begin{bmatrix} 3 & 4 \end{bmatrix}$  3.  $\mathbf{K} = \begin{bmatrix} 1 \\ 2 \end{bmatrix}$  4.  $\mathbf{K} = \begin{bmatrix} 3 \\ 4 \end{bmatrix}$ 

5. none of the above

## **Pole Placement**

With full-state feedback, the gains **K** can be adjusted to produce **ANY** set of *n* closed-loop poles!  $\rightarrow$  much more powerful than classical methods!

The design problem shifts ...

- from finding gains to optimize pole locations (classical view)
- to finding pole locations to optimize performance (modern view).

Examples: Next Lecture