

# Dynamic System Modeling and Control Design

## Frequency Response

Oct. 8, 2025

## Review: $s$ -Domain Description

- We describe an LTI system by

$$Y = H(s) U, \quad H(s) = \frac{n(s)}{d(s)},$$

where  $H(s)$  is a rational function of polynomials,

$$n(s) = b_{\hat{\ell}} s^{\hat{\ell}} + \dots + b_1 s + b_0, \quad d(s) = a_{\ell} s^{\ell} + \dots + a_1 s + a_0,$$

- using the heuristic rule

$$\frac{d}{dt} \longleftrightarrow s.$$

Turns a differential equation into algebra in the variable  $s$ .

# Natural Frequencies of Our System

In general,

$$(a_\ell s^\ell + \dots + a_1 s + a_0)Y = (b_{\hat{\ell}} s^{\hat{\ell}} + \dots + b_1 s + b_0)U.$$

Suppose  $u(t) = 0 \ \forall t \Rightarrow U = 0$  and  $y(t) = \sum_i \alpha_i e^{\lambda_i t}$ .

Then for each term,

$$a_\ell s^\ell Y \longleftrightarrow a_\ell \frac{d^\ell}{dt^\ell} y(t), \quad \frac{d^\ell}{dt^\ell} y(t) = \sum_i \lambda_i^\ell \alpha_i e^{\lambda_i t}.$$

$\alpha_i \neq 0$  iff  $\lambda_i$  is a root of  $d(s)|_{s=\lambda_i} = 0$

- The roots of  $d(s)$  are called the “natural frequencies.”

## Motivation: Why (Complex) Exponential Inputs?

We will choose an input function of the form:  $u(t) = \sum_i \beta_i e^{s_i t}$ .

- Many signals in engineering are oscillatory: vibrations, AC power, rotating machinery, etc.
- Because the system is linear and time-invariant; knowing the response to one sinusoid tells us how the system behaves for any sum of sinusoids.

Idea: We can probe our system with exponentials of our choosing,  $e^{s_0 t}$ , and analyze how the system output responds.

# Decomposing the System Response

- For any input,

$$y(t) = \underbrace{\sum_i \alpha_i e^{\lambda_i t}}_{\text{natural response}} + (\text{response due to an input}).$$

- If the natural frequencies  $\lambda_i$  have negative real parts, the natural response **decays**.
- Today's objective: understand the response due to a **sinusoidal input**.

# Complex Exponential In, Complex Exponential Out

Recall our setup:

$$(a_\ell s^\ell + a_{\ell-1} s^{\ell-1} + \cdots + a_1 s + a_0)Y = (b_{\hat{\ell}} s^{\hat{\ell}} + \cdots + b_1 s + b_0)U,$$

or equivalently,

$$d(s)Y = n(s)U.$$

We want to see how the system responds to an **exponential input**:

$$u(t) = e^{s_0 t}.$$

Because  $s \leftrightarrow \frac{d}{dt}$ ,

$$s^k U \longleftrightarrow \frac{d^k}{dt^k} e^{s_0 t} = s_0^k e^{s_0 t}.$$

**Try a particular solution:** assume  $y(t) = Ce^{s_0 t}$ , where  $C$  is a complex constant to be found.

# Complex Exponential In, Complex Exponential Out

Substitute  $y(t) = Ce^{s_0 t} \longleftrightarrow Y$  and  $u(t) = e^{s_0 t} \longleftrightarrow U$  into

$$d(s)|_{s=s_0} Y = n(s)|_{s=s_0} U.$$

Then:

$$d(s_0) Ce^{s_0 t} = n(s_0) e^{s_0 t}.$$

If  $d(s_0) \neq 0$ , we can cancel  $e^{s_0 t}$  to obtain

$$C = \frac{n(s_0)}{d(s_0)}, \Rightarrow \boxed{y(t) = \frac{n(s_0)}{d(s_0)} e^{s_0 t}}.$$

## Special case: Complex exponential input.

If  $d(s_0) \neq 0$ , we can cancel  $e^{s_0 t}$  to obtain

$$C = \frac{n(s_0)}{d(s_0)}, \Rightarrow \boxed{y(t) = \frac{n(s_0)}{d(s_0)} e^{s_0 t}}.$$

Let  $s_0 = j\omega$ :

$$H(j\omega) \equiv \frac{n(j\omega)}{d(j\omega)}, \quad y(t) = H(j\omega) e^{j\omega t}.$$

- Recall Euler's identity:  $e^{j\theta} = \cos \theta + j \sin \theta$
- The complex exponential  $e^{j\omega t}$  NEVER decays to zero.

# Transient and Steady-State Responses

- For a persisting sinusoidal input,

$$y(t) = \underbrace{\sum_i \alpha_i e^{\lambda_i t}}_{\text{natural response}} + (\text{sinusoidal steady state response}).$$

- If the natural frequencies  $\lambda_i$  have negative real parts, the natural response **decays**.
  - Dies out with time; referred to as the **transient** response.
- We have the tools necessary to determine the **sinusoidal steady state** response.

# Sinusoidal Steady State

Suppose our input takes the form  $u(t) = \cos \omega t$ . What is  $y(t)$ ?

Euler's identity:  $u(t) = \frac{1}{2} (e^{j\omega t} + e^{-j\omega t})$

- Through superposition, we know that,

$$\begin{aligned} y(t) &= \frac{1}{2} (H(j\omega)e^{j\omega t} + H(-j\omega)e^{-j\omega t}), \\ &= \Re \{ H(j\omega)e^{j\omega t} \}, \\ &= \Re \{ |H(j\omega)| e^{j\angle H(j\omega)} e^{j\omega t} \}, \\ &= \Re \{ |H(j\omega)| e^{j(\omega t + \angle H(j\omega))} \}, \\ &= |H(j\omega)| \cos(\omega t + \angle H(j\omega)), \end{aligned}$$

- Compared with  $u(t)$ , output  $y(t)$ 's magnitude scales by  $|H(j\omega)|$  and its phase changes by  $\angle H(j\omega)$ .

# Defining the Frequency Response

- The **frequency response** of a continuous-time LTI system is

$$H(j\omega) \equiv H(s)|_{s=j\omega}.$$

- $H(j\omega) = |H(j\omega)|e^{j\angle H(j\omega)}$  is complex.
- $|H(j\omega)|$ : amplitude ratio (gain)  $\angle H(j\omega)$ : phase shift.
- Together, they describe the steady-state output for any sinusoidal input.

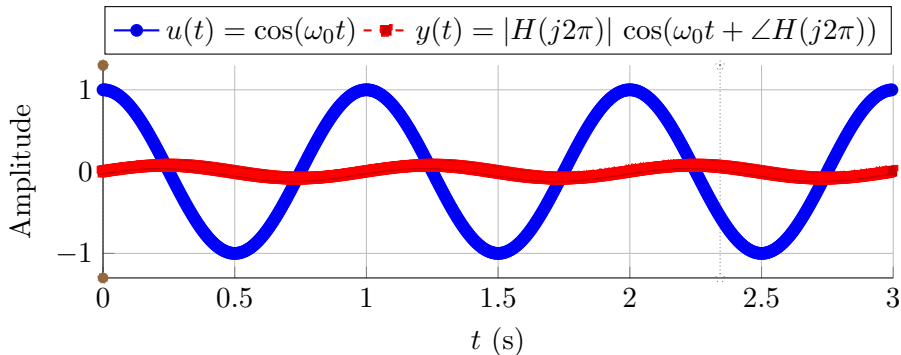
# Interpreting the Frequency Response

- $|H(j\omega)|$ : how much the system amplifies or attenuates each input frequency.
- The shape of  $|H(j\omega)|$  reveals which input frequencies excite the system strongly (those near its natural frequencies).
- $\angle H(j\omega)$ : how much the system shifts each frequency in time.
- The frequency response is therefore the bridge between the system's natural dynamics and its steady-state behavior under sinusoidal forcing.

Example:  $H(s) = \frac{1}{1 + 2s}$  for single input

Suppose  $H(s) = \frac{1}{1 + 2s}$ . We input  $u(t) = \cos(2\pi t)$ .

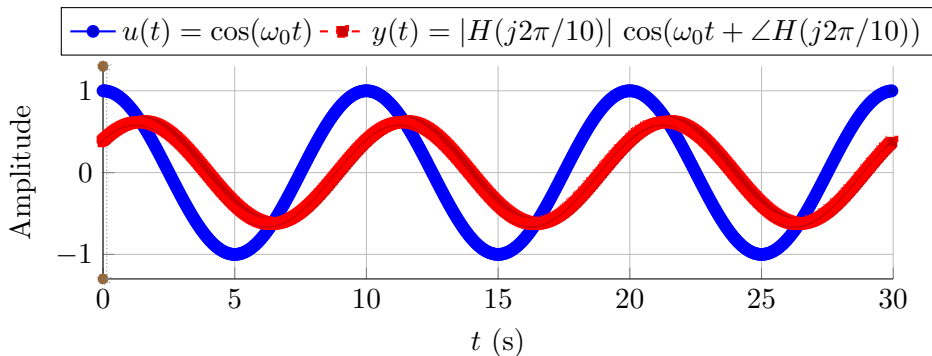
- $|H(j2\pi)| \approx 0.079$
- $\angle H(j2\pi) \approx -1.49$  rad



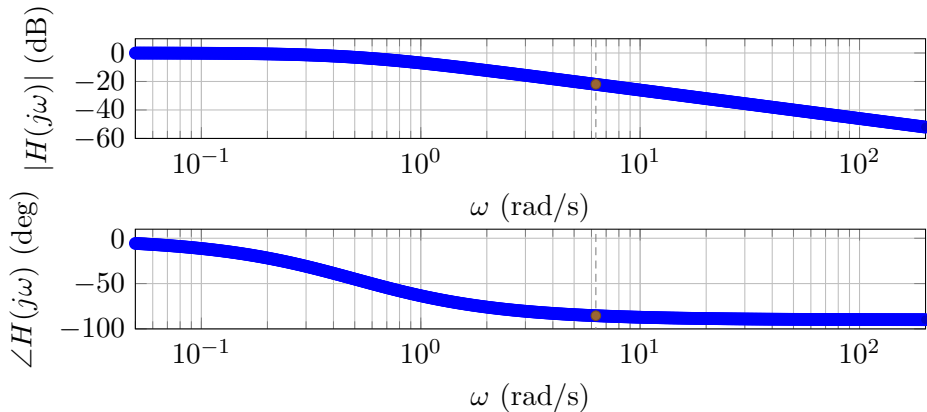
Example:  $H(s) = \frac{1}{1 + 2s}$  for single input

Suppose  $H(s) = \frac{1}{1 + 2s}$ . We input  $u(t) = \cos(\frac{2\pi}{10}t)$ .

- $|H(j2\pi/10)| \approx 0.625$
- $\angle H(j2\pi/10) \approx -0.90$  rad



Example:  $H(s) = \frac{1}{1 + 2s}$  for different inputs



At  $\omega_0 = 2\pi$ :  $|H(j\omega)| \approx -22.0$  dB,  $\angle H(j\omega) \approx -85.4^\circ$ .

We will learn more about generating these plots next week.

# Summary

- Natural frequencies: describe the system's own free oscillations (transients).
- Frequency response:  $H(j\omega)$  — the system's gain and phase shift for forced sinusoidal inputs.
- The total response:

$$y(t) = \underbrace{\sum_i \alpha_i e^{\lambda_i t}}_{\text{transient (natural)}} + \underbrace{|H(j\omega)| \sin(\omega t + \angle H(j\omega))}_{\text{steady-state (forced)}}.$$

- As transients decay, the steady-state sinusoid remains.
- $H(j\omega)$  fully characterizes this steady-state relationship.