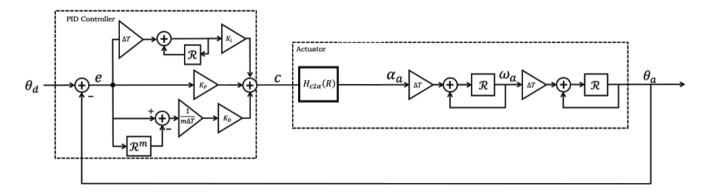
# Code of Arms: Post-Lab

This post-lab is due on Thursday, March 9, 2023, at 11:59pm EST.

## **Problems**

#### 1. System (Transfer) Function

In Lab 2 (Code of Arms), we studied a propellor arm system that is controlled by a PD or PID controller. The system block diagram is shown below:



Determine the system function (aka transfer function) for this block diagram as follows. Express your answers in terms of the operator  $\mathcal{R}$ .

1a. Let  $C = K(\mathcal{R})E$ . Determine an expression for  $K(\mathcal{R})$  in terms of the controller constants  $K_p$ ,  $K_d$ ,  $K_i$ , and m as well as the step size  $\Delta T$ .

$$K(\mathcal{R}) =$$

1b. Let  $A_a = H_{c2a}(\mathcal{R})C$ . Determine an expression for  $H_{c2a}$  for the "Non-Instant" motor model from part 7 of the lab.

$$H_{c2a}(\mathcal{R}) =$$

1c. Let  $\Theta_a = H_{a2\theta}(\mathcal{R})A$ . Determine  $H_{a2\theta}$ .

$$H_{a2\theta}(\mathcal{R}) =$$

1d. Let  $\Theta_a = H_c(\mathcal{R})\Theta_d$ , where  $H_c(\mathcal{R})$  represents the "closed-loop" transfer function. Determine  $H_c(\mathcal{R})$  in terms of  $K(\mathcal{R})$ ,  $H_{c2a}(\mathcal{R})$ , and  $H_{a2\theta}(\mathcal{R})$ .

$$H_c(\mathcal{R}) =$$

### 2. Step Responses

Compare the step responses of the PD and PID systems as follows. Set the input  $\theta_d[n]$  to a unit step

$$\theta_d[n] = \begin{cases} 1 & \text{if } n \ge 0\\ 0 & \text{otherwise} \end{cases}$$

Then use your favorite computer language to compute the step response under the following conditions. 2a. PD controller. Set  $K_i = 0$  and use the control parameters that you used in lab. Paste a plot of the

step response in the box below.

What is the steady-state error of this system?

 $steady-state\ error=$ 

2b. PID controller. Now set  $K_i$  to the value you used in lab and calculate the response with a PID

controller.

What is the steady-state error of this system?

steady-state error =

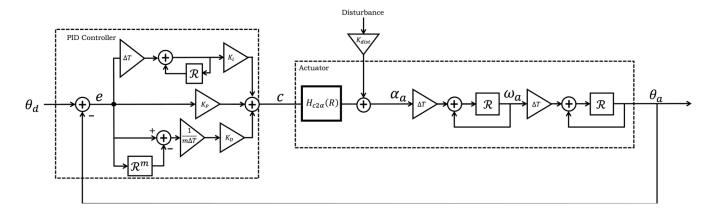
#### 3. Disturbance Measurements

From a modeling point of view, adding the integral term doesn't help much for this system. However, the integral term helps significantly when dealing with disturbances. You have collected data when a weight is dropped onto the propellor arm. Let's compare the disturbance rejection properties for the PD and PID controllers.

| PD and PID controllers.  |  |  |  |  |
|--|--|--|--|--|
| 3a. Show the experimental control of the second control of the sec | nental data of your PD controller when a weight is added.  |  |  |  |
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| 3b. Show the experimental 3b.  | nental data of your PID controller when a weight is added. |  |  |  |
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| 3c. Which controller   | provides better disturbance rejection?                     |  |  |  |
|  |  |  |  |  |
| PD or PID:   |  |  |  |  |
|  |  |  |  |  |

## 4. Disturbance Model

We can model the effect of the disturbance by adding a new input pathway as shown below.

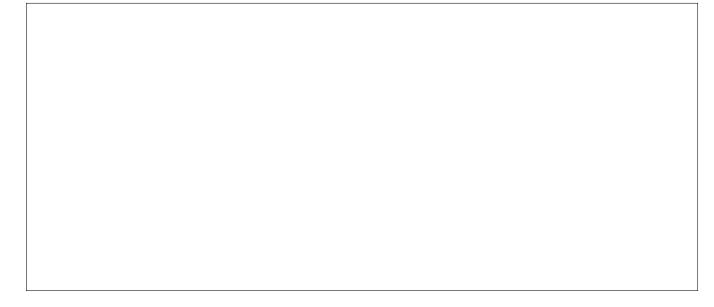


Determine the disturbance transfer function  $H_{dist}$  as follows

$$\Theta_a = H_{dist}(\mathcal{R}) M_{dist}$$

where  $M_{dist}$  represents the disturbance input. When you write the disturbance transfer function, you can set the other input  $\Theta_a$  to 0.

4a. Derive the disturbance transfer function.



|     | sturbance rejection with a PD controller and PID controller. In the simulations, you copping a weight as a step function. Use $K_{dist} = 10$ . | an | mode |
|-----|---|----|------|
|     | . Disturbance rejection with a PD controller.   |    |      |
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| 4c. | . Disturbance rejection with a PID controller.  |    |      |
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Implement this disturbance function in your favorate programming language. Simulate the system