Second order DT system, Proportional control, and PD control

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Outline:
1. Second order DT system: line following example
2. Stability of a second order DT system under a proportional controller
3. Proportional derivative controllers

1. Second order DT system: line following example

Thus far, we studied first order systems in the previous lectures. We are going to use an example to study second order systems, where new stability properties arise and the proportional controller becomes insufficient. Let’s consider a line following example illustrated below.

\[
\begin{align*}
d[n] &= d[n-1] + \Delta T \sin \theta[n-1] \\
\theta[n] &= \theta[n-1] + \Delta T \omega[n-1] \\
\omega[n] &= \gamma u[n]
\end{align*}
\]

Suppose the robot wants to move along a straight line. The robot has a constant velocity \( V \), and we can control its rotation speed \( \omega[n] \) through an input \( u[n] \):

\[
\omega[n] = \gamma u[n]
\]

In this problem, we have an optical sensor that can measure the distance between the robot and the line. Our goal is to design a controller that minimizes the distance between the desired position \( d_d[n] \) (line) and the measured position \( d_m[n] \).

The discrete time kinematic equation is given by:

\[
d_m[n] = d_m[n-1] + \Delta T V \sin \theta[n-1]
\]

This is a nonlinear equation, and we need to linearize the term \( \sin \theta \). We have \( \sin \theta \approx \theta \) for small \( \theta \). The system equation becomes:

\[
d_m[n] = d_m[n-1] + \Delta T V \theta[n-1]
\]

We need to write \( \theta[n] \) in terms of \( d_m[n] \), where \( \theta[n] \) is given by:

\[
\theta[n] = \theta[n-1] + \Delta T \omega[n-1] = \theta[n-1] + \Delta T \gamma u[n-1]
\]

Since we can measure the distance \( d_m[n] \), we can set up a proportional controller relative to the measured distance:
\[ u[n] = K_p(d_d[n] - d_m[n]) \]

Substituting this controller into our system equation, we obtain:
\[ \theta[n] = \theta[n - 1] + \Delta TK_p\gamma(d_d[n - 1] - d_m[n - 1]) \]

Now let's write the system equations and simplify:
\[ d_m[n] = d_m[n - 1] + \Delta TV\theta[n - 1] \]
\[ d_m[n - 1] = d_m[n - 2] + \Delta TV\theta[n - 2] \]

Subtracting these two equations, we obtain:
\[ d_m[n] - d_m[n - 1] = d_m[n - 1] - d_m[n - 2] + \Delta TV(\theta[n - 1] - \theta[n - 2]) \]

Next, we substitute the difference of \( \theta \):
\[ d_m[n] - d_m[n - 1] = d_m[n - 1] - d_m[n - 2] + \Delta TV(\Delta TK_p\gamma(d_d[n - 2] - d_m[n - 2])) \]

We can simplify this equation, collect terms, and get the control system equation:
\[ d_m[n] - 2d_m[n - 1] + (1 + \Delta T^2VK_p\gamma)d_m[n - 2] = \Delta T^2VK_p\gamma d_d[n - 2] \]

Note that the variable in this equation is \( d_m \), and we have the indices \( n \), \( n-1 \), and \( n-2 \). This is a 2\textsuperscript{nd} order DT system with proportional control.

2. **Stability of a second order DT system under a proportional controller**

We need to go through the same exercise again to analyze the behavior of a 2\textsuperscript{nd} order DT system. The general solution of a 2\textsuperscript{nd} order DT system with zero-driving (\( d_d[n] = 0 \)) is given by:
\[ d_m[n] = C_1\lambda_1^n + C_2\lambda_2^n \]

where \( \lambda_1 \) and \( \lambda_2 \) are natural frequencies, and \( C_1 \) and \( C_2 \) are coefficients determined by the initial conditions. To analyze system stability, we need to solve for the value of \( \lambda_1 \) and \( \lambda_2 \). We can substitute the solution \( d_m[n] = \lambda^n \) into the system equation and then solve for \( \lambda \). Here we are interested in the homogeneous solution (when the right hand driving function is 0). We have:
\[ \lambda^n - 2\lambda^{n-1} + (1 + \Delta T^2VK_p\gamma)\lambda^{n-2} = 0 \]
\[ \lambda^2 - 2\lambda + (1 + \Delta T^2VK_p\gamma) = 0 \]

We can solve for \( \lambda \) and obtain:
\[ \lambda = 1 \pm j\sqrt{\Delta T^2VK_p\gamma} \]

This is an interesting result that we should carefully study. First, \( \lambda \) is a complex number, which contains a real part and an imaginary part. We see that both have a larger than 1 amplitude. For this line following example under proportional control, the system is unstable regardless of the
$K_p$ values we pick! This is an important result. For higher order systems, a simple proportional controller usually does not work well. Implementing a proportionally controller can lead to an unstable system. The sketch below illustrates a sample line-following experiment.

Another important concept that we want to introduce is called root locus plot. It is a plot in the complex plane that shows the location of natural frequencies as a function of our controller parameter. In this example, the only controller parameter we have is $K_p$. How does a change of $K_p$ changes the two $\lambda$ values? This plot is shown below, where we start with $K_p = 0$ and end with $K_p \to \infty$. Note that all $\lambda$ values are outside of the unit circle, which means the system is unstable. For discrete time problems, the unit circle marks the stability region. For a given parameter value, if all the associated $\lambda$ values are within the unit circle, then the system is stable. Otherwise, the system is unstable.

The root locus plot shows proportional control of this second order system is unstable. This seems to be a major problem. How can we stabilize a higher order system and then optimize the controller parameters according to some metrics (fastest convergence or smallest steady state error)? Next, we will introduce the proportional-derivative (PD) controller.

3. **Proportional-derivative (PD) controller and 3rd order system**

While a proportional (P) controller cannot stabilize the 2nd order system, we can implement a proportional-derivative (PD) controller. The intuition is that our controller should not only care about how far the car is relative to the setpoint, but also about the rate of change. The PD controller is given by:

$$c[n] = K_p(d_d[n] - d[n]) + K_d\left(\frac{d_d[n] - d_d[n-1]}{\Delta T} + \frac{d[n] - d[n-1]}{\Delta T}\right)$$

If we substitute this controller into the system equation, we will obtain:
\[ d[n] - 2d[n-1] + d[n-2] = \Delta T^2 V_y [K_p (d_d[n-2] - d[n-2]) + K_d \left( \frac{d_d[n-2] - d_d[n-3]}{\Delta T} \right) + \frac{d[n-2] - d[n-3]}{\Delta T}] \]

We can rearrange this equation and obtain:
\[
d[n] - 2d[n-1] + d[n-2](1 + \Delta T^2 V_y K_p + K_d V_y \Delta T) + d[n-3](-K_d V_y \Delta T) = d_d[n-2](\Delta T^2 V_k + K_d \Delta T V_y) + d_d[n-3](-K_d \Delta T V_y)
\]

This is a 3\textsuperscript{rd} order difference equation, and it has the following solution:
\[ d[n] = C_1 \lambda_1^n + C_2 \lambda_2^n + C_3 \lambda_3^n \]

Now there are 3 natural frequencies, and they are functions of Kp and Kd. To design a good controller, we need to choose Kp and Kd such that our system is stable. Since the equation is sufficiently complex, we will use a numerical tool to generate the root locus plots.

Case 1: set Kd = 0, V=1, \(\gamma = 1\), \(\Delta T = 0.01\), and vary Kp. This becomes a proportional controller with two natural frequencies. We see that the system is unstable regardless of the Kp value we choose.

Case 2: set Kd = 20, V=1, \(\gamma = 1\), \(\Delta T = 0.01\), and vary Kp. We see that now there is an optimal Kp value that corresponds to the fastest convergence. In the next lecture, we will introduce more MATLAB tools for analyzing a control system.